

## **Model Sizes in Implicit and Explicit Calculations**

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## Element Sizes in implicit and explicit calculation

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## Agenda

- Tecosim-best partner for simulation
- Introduction
- Different Mesh sizes in complex structures
- Analysis of Results/ Conclusion
- Outlook

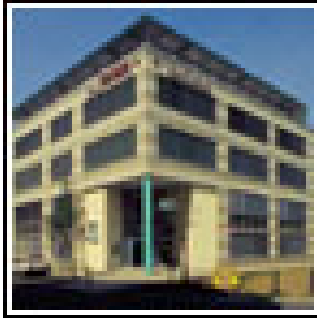
## The Company

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- Company founded in 1992
- Strategic CAE Partner of Major OEM's and Suppliers
- Core Competence in CAE
- Turnover € 8,5 Mio in 2005
- Currently more than 150 Employees at 5 European Locations (nearly all with university engineering degree, 10 PhD's)
- State-of-the-Art IT Infrastructure
- Co-operation with Leading Engineering and Service Supplier
- Comprehensive liability insurance
- Quality Management System according to ISO 9001, UM 14001 and Customer Specific Requirements (FORD Q1 Award)

# Locations & References



Rüsselsheim



Köln

- Audi AG
- Adam OPEL AG
- Daimler Chrysler
- Daihatsu
- Fiat
- FORD
- General Motors
- HONDA
- Hyundai
- ISUZU
- KIA
- John Deere
- Jaguar
- Landrover
- Nissan
- PORSCHE AG
- Toyota
- AMG
- Autoliv
- Bayer AG
- Bentler
- Bertone
- Bosch/ Blaupunkt
- Degussa-Hüls AG
- Dynamit Nobel
- EADS
- Faurecia
- Getrag
- Hella KG
- Johnson Controls
- Karmann
- Lear
- Magna
- Thyssenkrupp
- TRW Automotive
- Mahle
- MAN
- Mannesmann/Sachs
- Siemens VDO
- Wagon Automotive



Leonberg

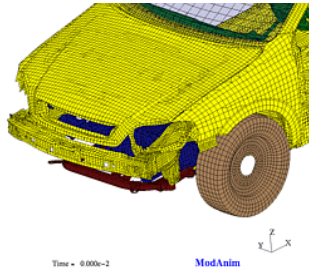


Basildon (UK)



Coventry (UK)

# CAE Portfolio

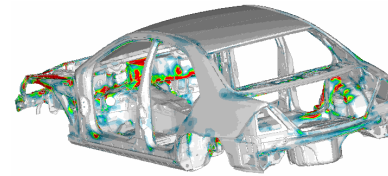


CRASH

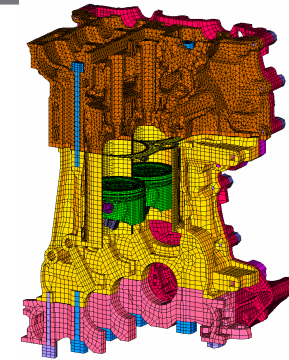
Frontcrash  
Time: 0



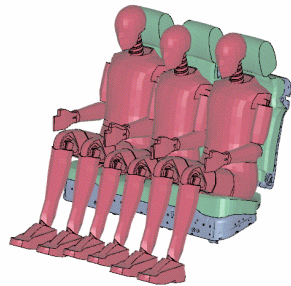
Safety



NVH / Durability

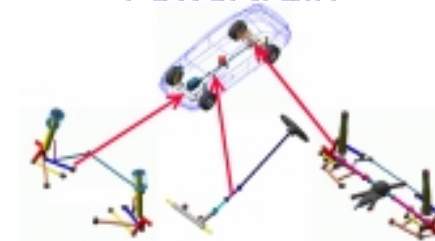


Powertrain

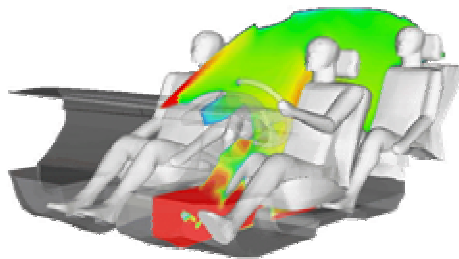


Seats

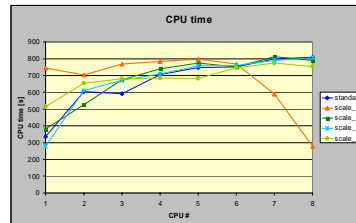
## CAE Portfolio



Multi Body Systems (MBS)



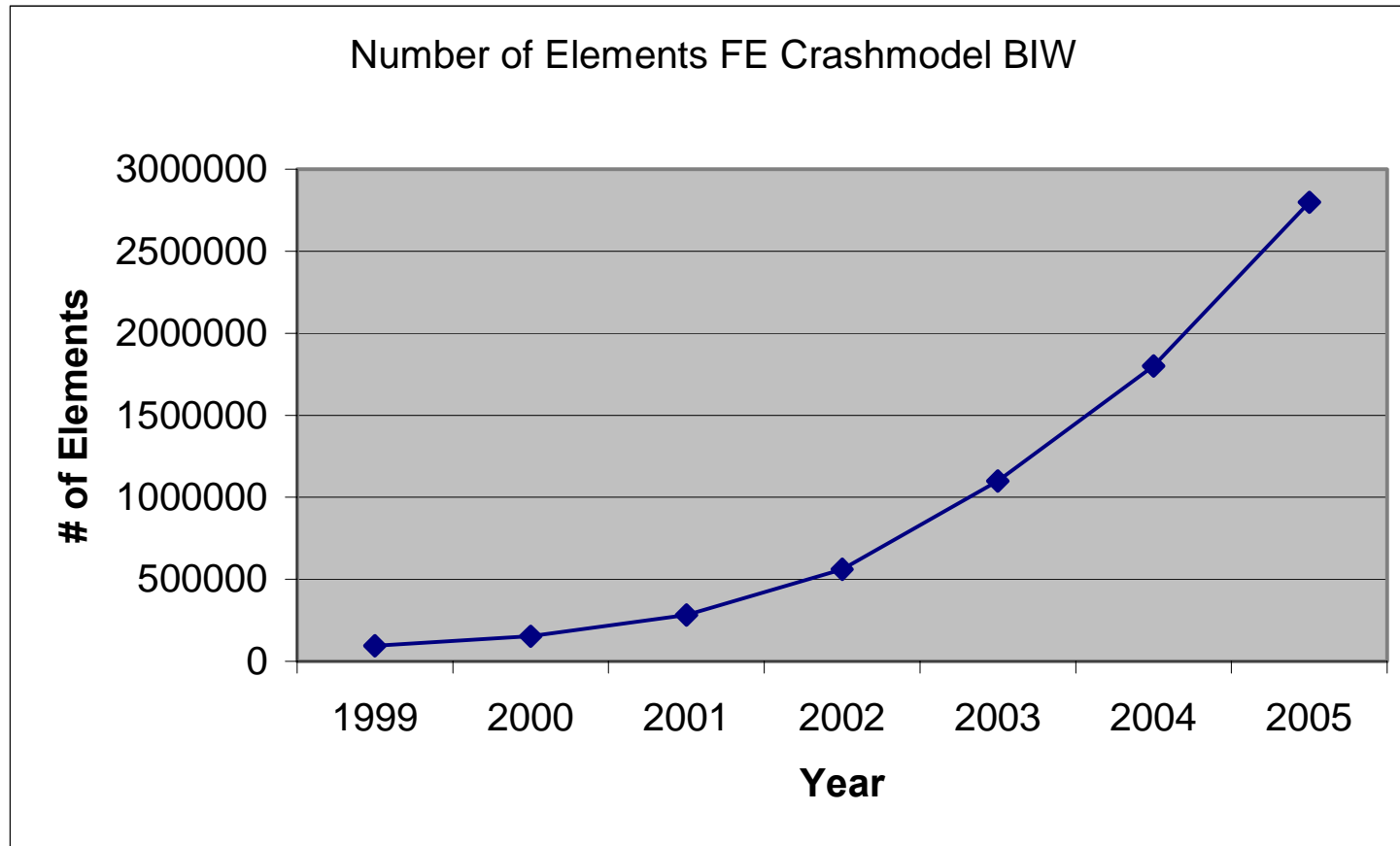
CFD



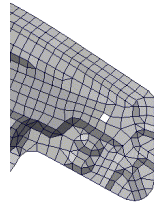
Optimization

**TECIODM**  
best software for simulation  
**TECPROM**  
best process for simulation  
**TECBENCH**  
best benchmarking for simulation

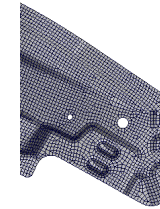
CAE Products



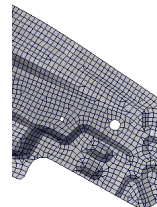
- How should a mesh look like?



**TEC|ODM** Mesh Element size 12 mm



**TEC|ODM** Mesh Element size 4 mm



**TEC|ODM** Mesh Element size 6 mm



## Simple box crash experiment:

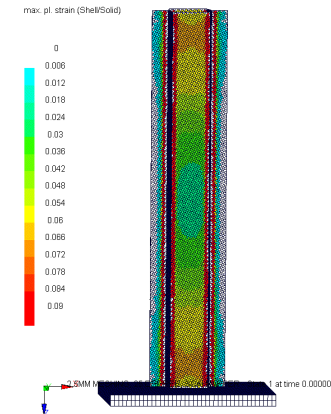
Box section 50 mm x 80 x 500mm,  $t = 1.0\text{mm}$ , mild steel

Varied parameters:

- average edge length 15/10/5/2,5mm
- mesh orientation 0deg/ 25deg
- different mesh/ integration method: Belytschko-Tsay/ Fully Integration
- Varied number of spotwelds
- With and without mapping or stamping data
- Renumbering and move in space

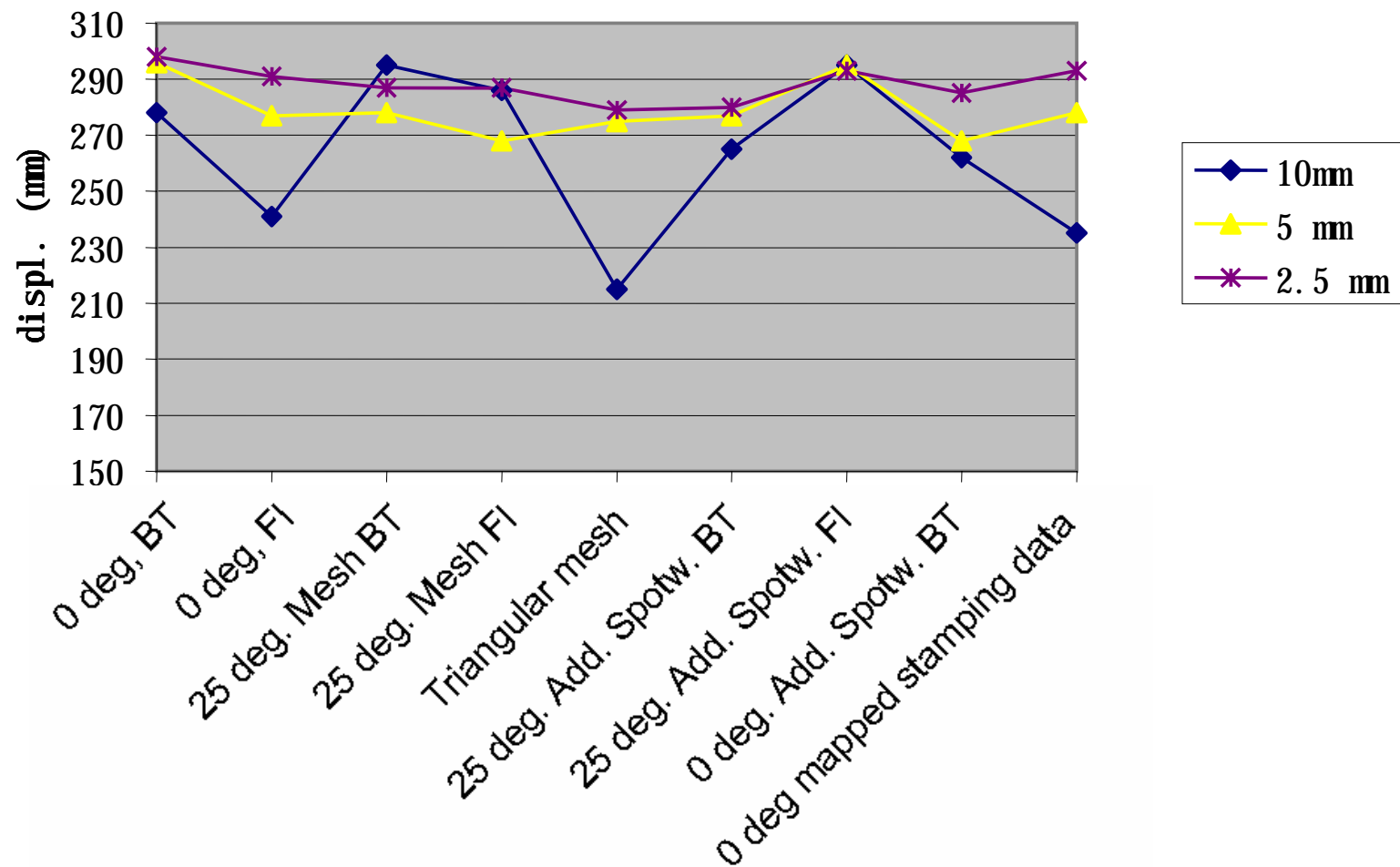
Objective:

- Is the result depending on the element length?
- Is the result depending on the element orientation?
- How does mapping influence/stabalise the results?
- How do small changes in the input influence the results?



## Analysis: comparison max displacement variation

variation for different mesh sizes

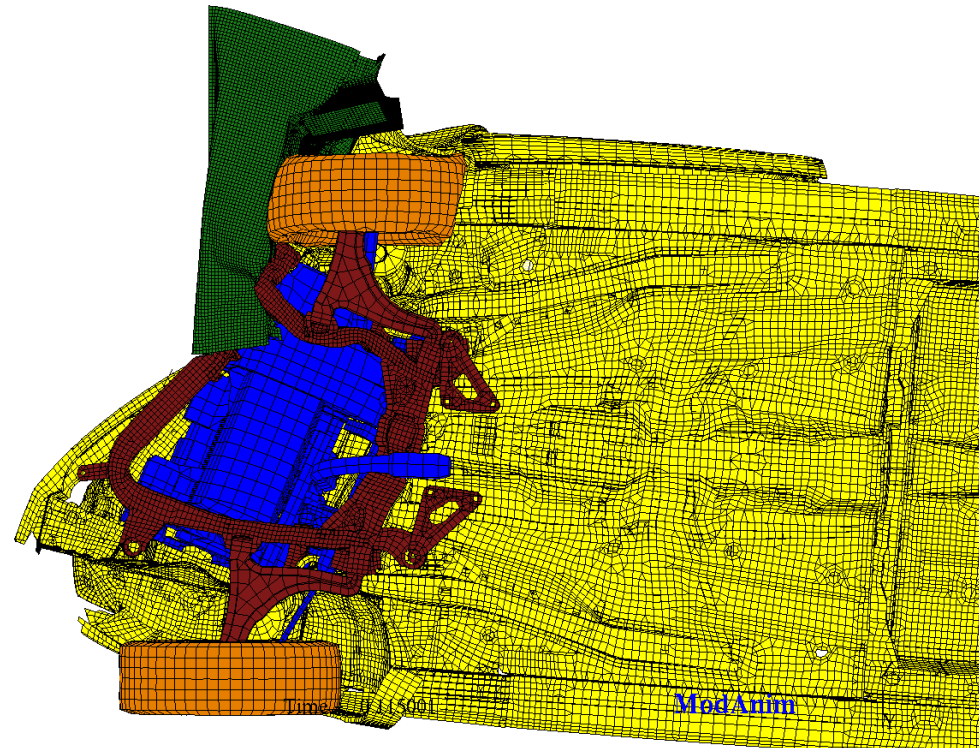


## Conclusion

- If you know the collapse mode of a part you can use a coarse mesh which should be orthogonal in the collapse direction (so you can achieve “superconvergence”)
  - If you doesn't know the collapse mode of a part; Please use finer meshes
  - No one knows the exact collapse mode of all the parts in a vehicle!
  - Meshing rules for orthogonal /Mapping/Integration schemes meshes are important for coarser meshes but not important for finer meshes.
- 
- *Creation of finer meshes can be automated by **TEC|ODM!***

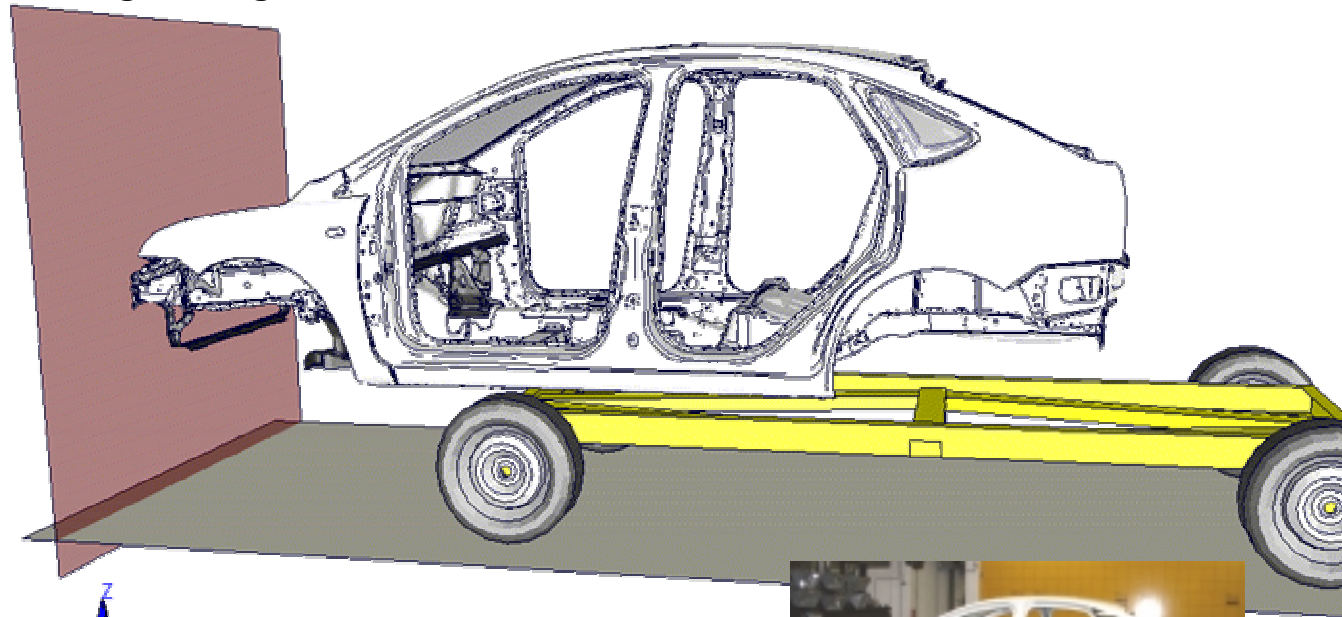
## Outlook

The crashbox sample will be applied to a complete vehicle to find out about the time saving potential and the influence on the results.



## Front end experiment:

Complete car frontend in crash test condition:  
7,8m/s, 1710 kg vs rigid wall



Objective:



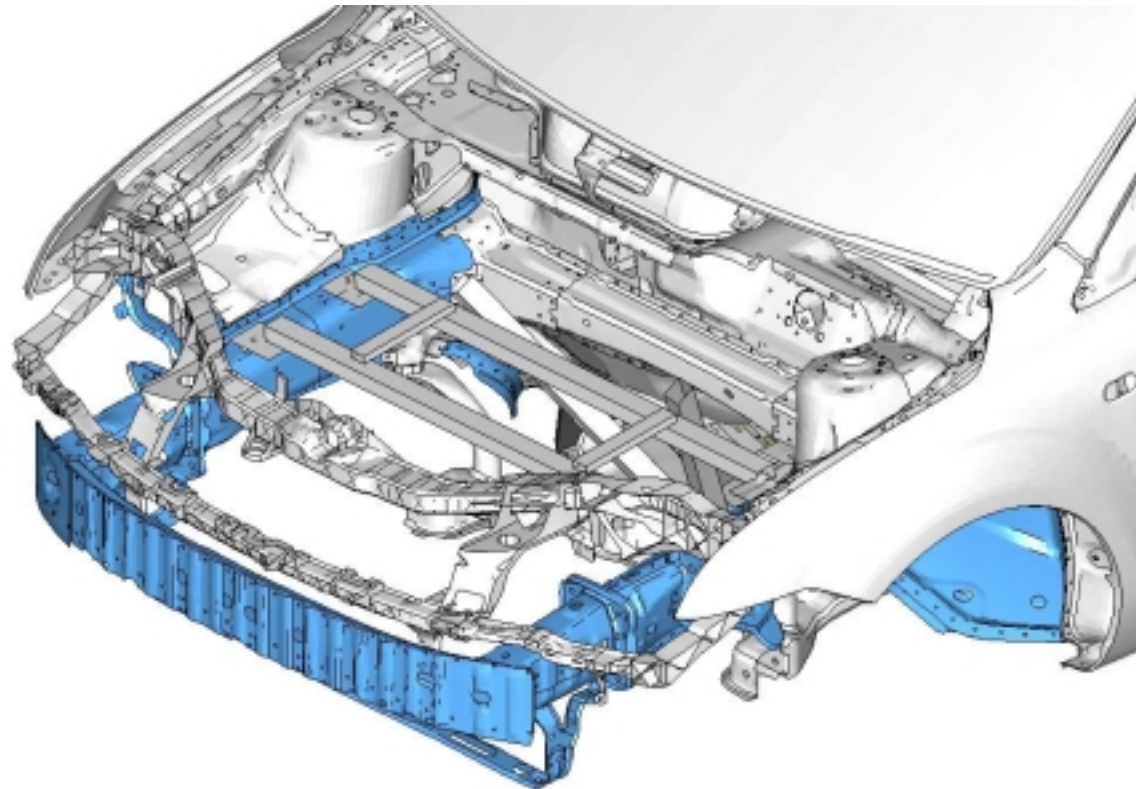
BMW 100% OFFSET

- Is the result depending on the element length?
- How do small changes in the input influence the results?

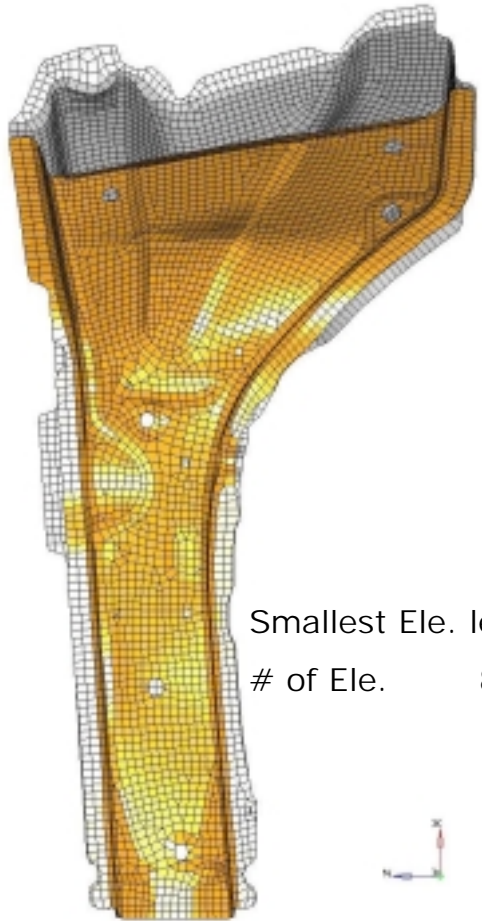


## modified **TEC|BENCH™** model

- Refinement of Bumper, Crashbox, side member, etc

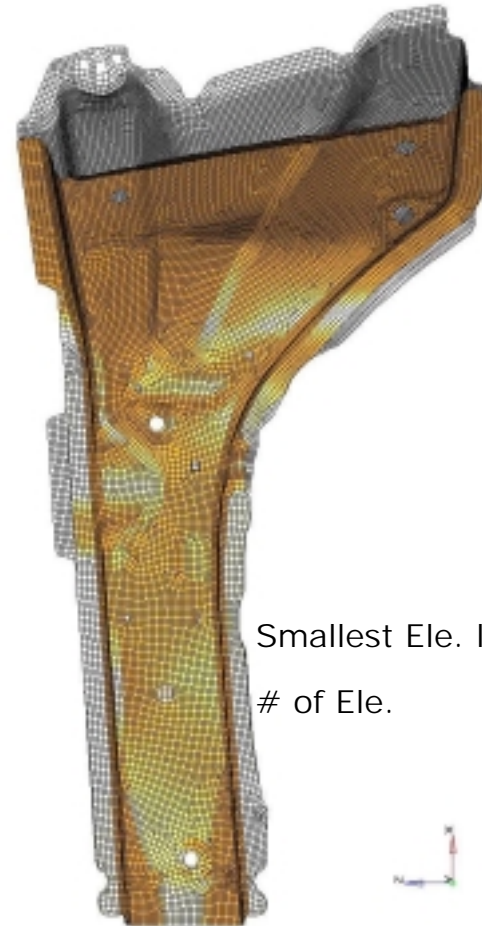


Example of refinement of **TEC|BENCH™** model



Smallest Ele. length 6,21 mm  
# of Ele. 8.588

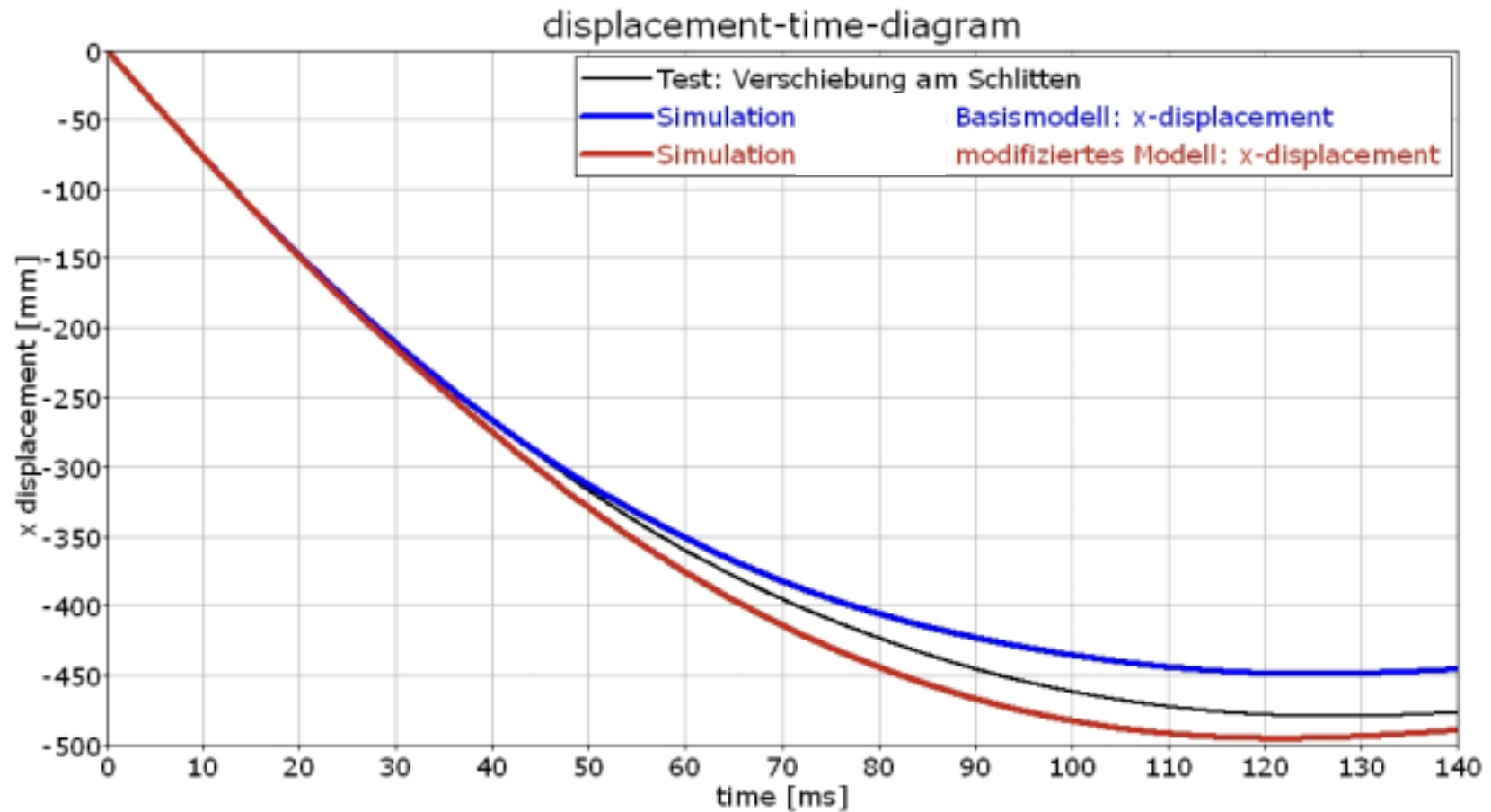
Base model



Smallest Ele. length 3,11 mm  
# of Ele. 34.119

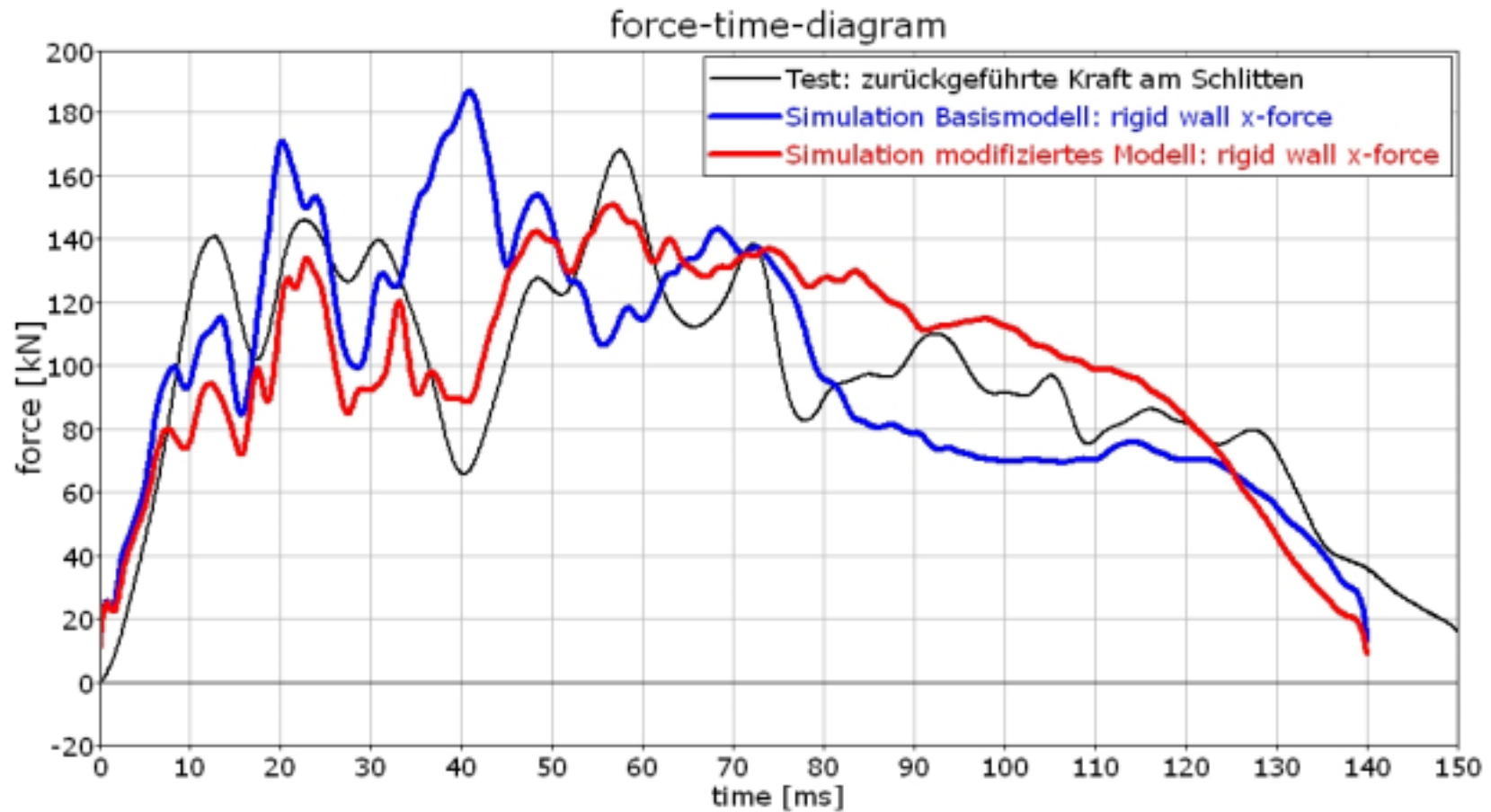
modified model

## Comparison Test/rough model/fine model

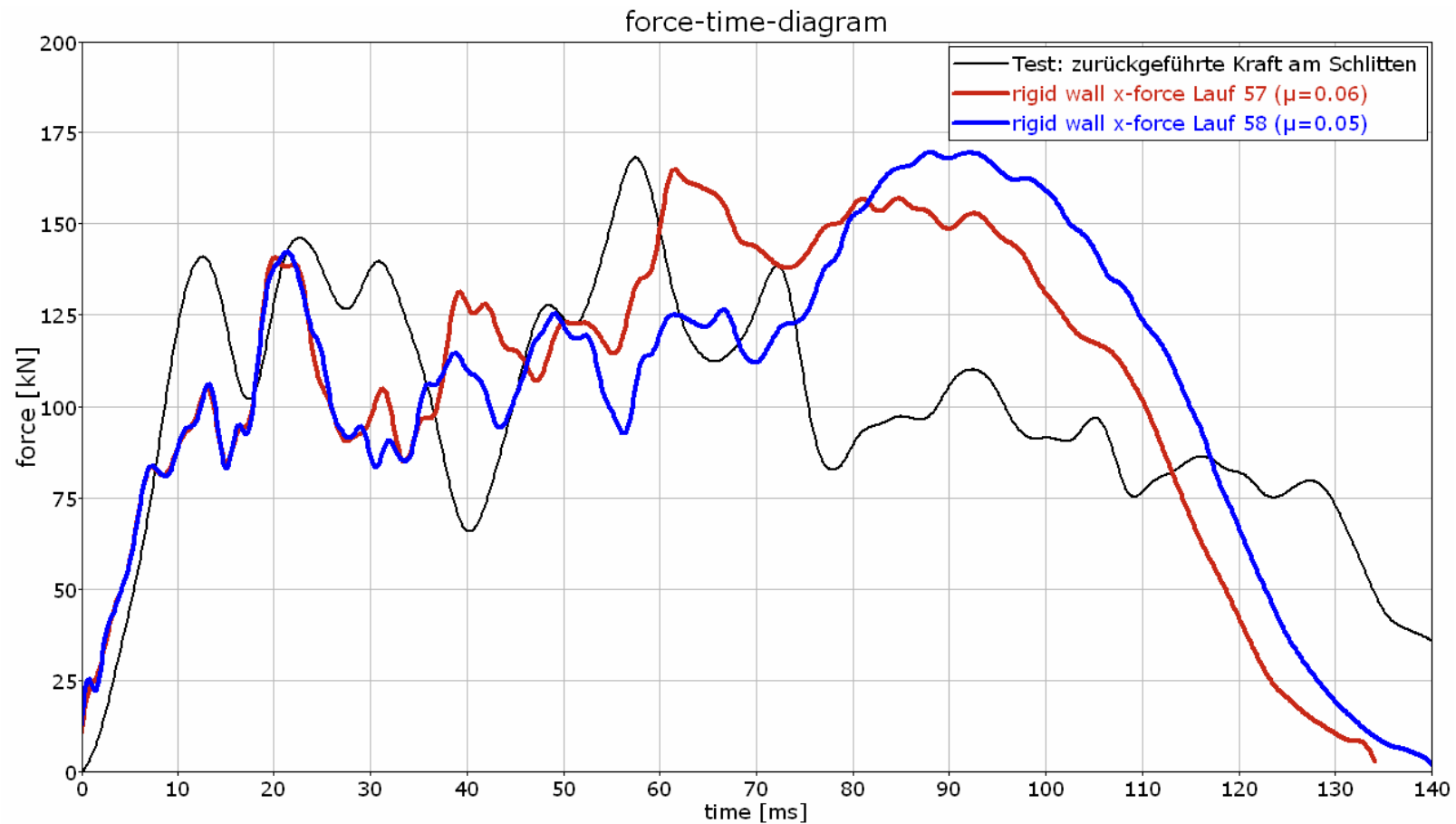




## Comparison Test/rough model/fine model



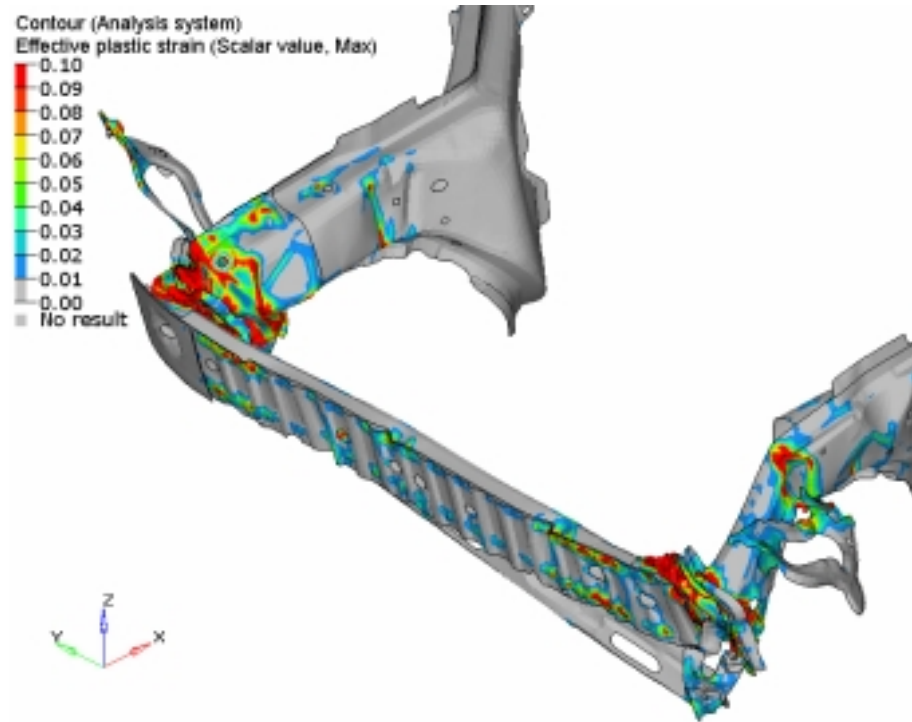
## Sensitivity check: Test/rough model with changed rigid wall friction ( $\mu = 0,05$ + $\mu = 0,06$ )



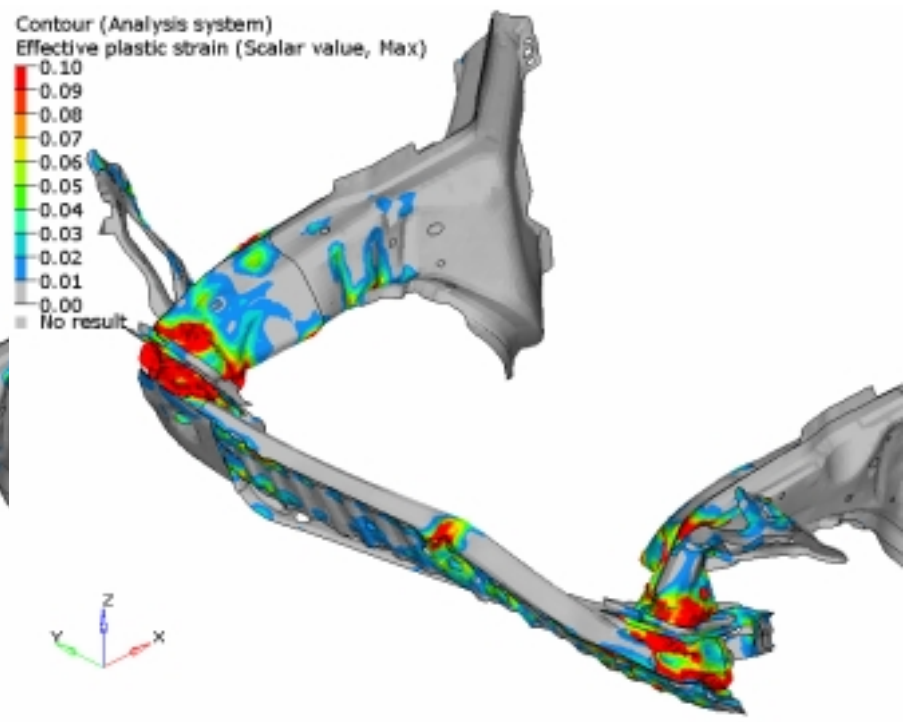
# Different Mesh sizes in complex structures



## Comparison rough model/fine model



Base model



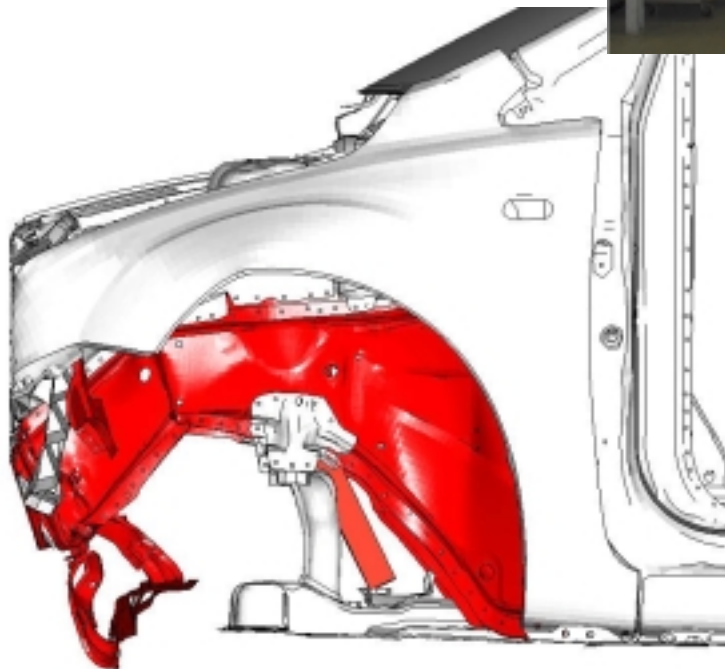
modified model

Different Mesh sizes in complex structures

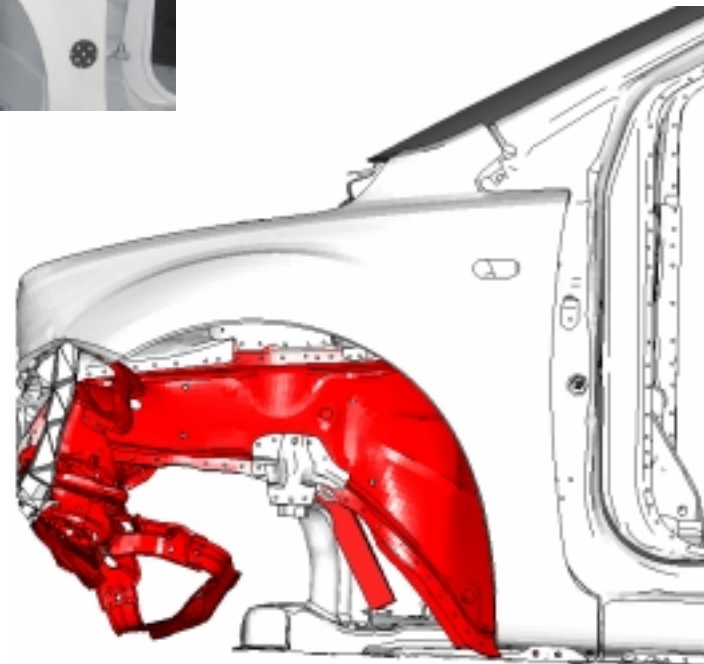


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## Comparison Test/rough model/fine model



Base model

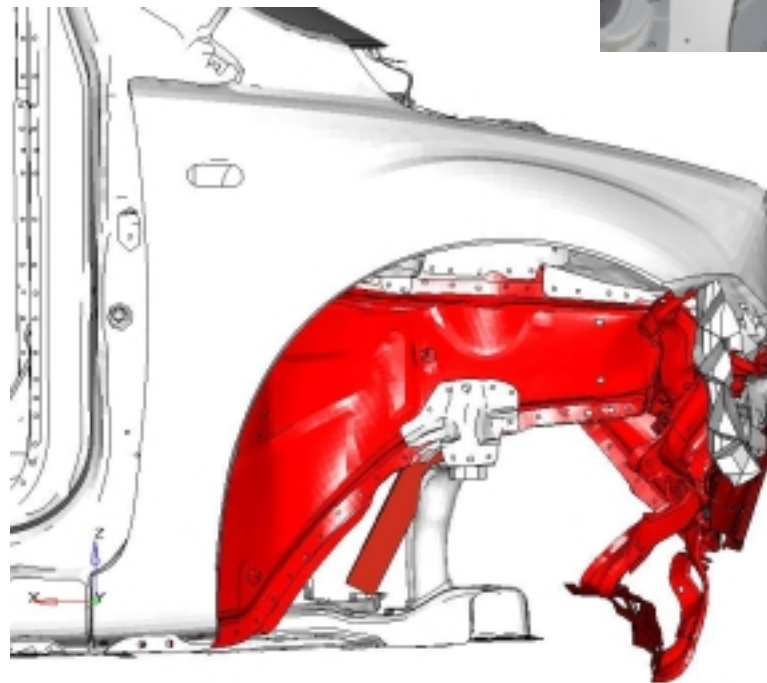


modified model

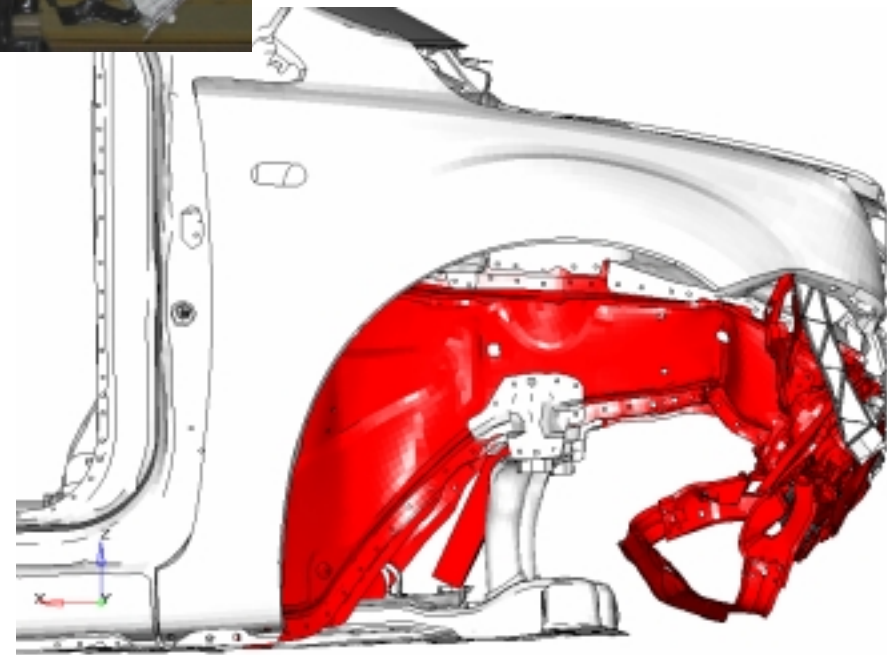
Different Mesh sizes in complex structures



## Comparison Test/rough model/fine model



Base model



modified model

## Influence in NVH global modes and frequencies

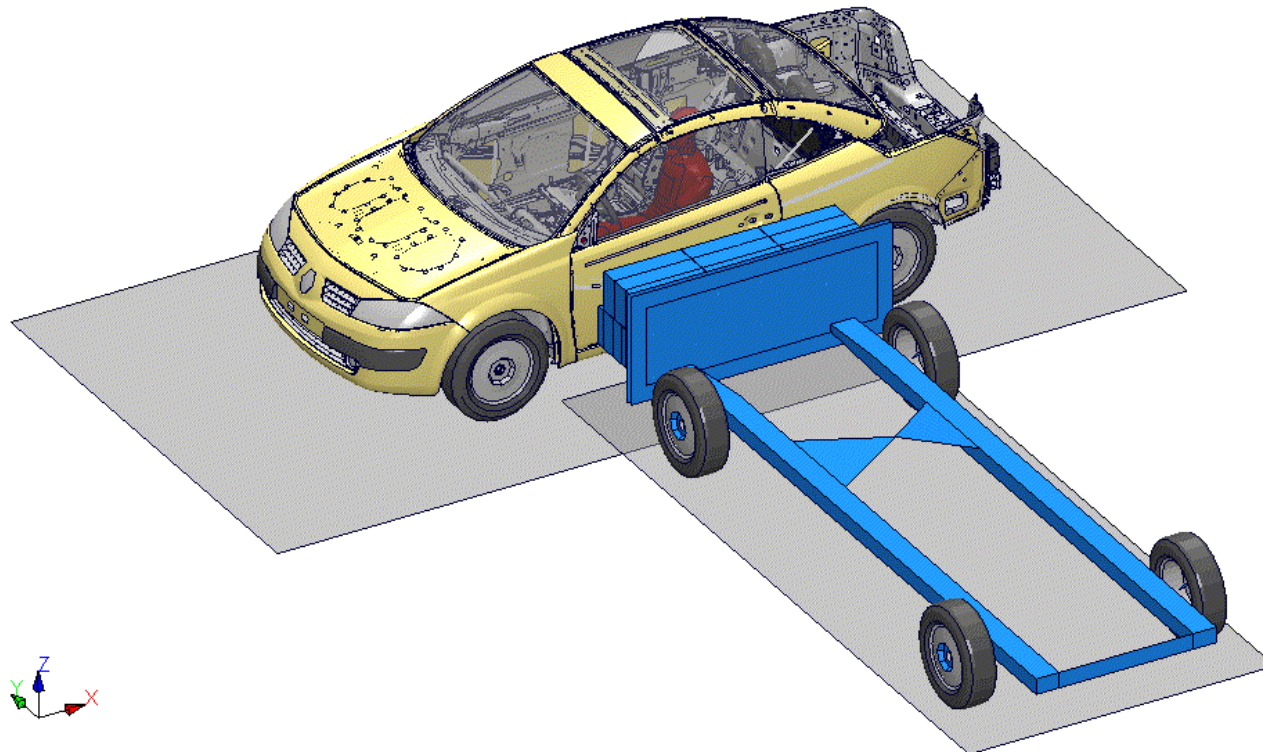
mode test	frequency [Hz]	description	mode sim.	frequency [Hz]	modal weight [kg]	coarse model		fine model	
						MAC	rel. frequency	MAC	rel. frequency
1	35,61	global torsion	1	34,28	64,00	97%	96%	97%	97%
2	37,78	normal upper window mode	3	38,77	5,68	91%	103%	91%	102%
3	42,28	lateral bending	2	36,62	38,30	85%	87%	87%	86%
4	46,03	vertical bending	4	43,60	0,49	88%	95%	83%	93%
5	48,55	Oscillating body mode	5	45,16	1,60	63%	93%	62%	91%
6	52,22	2nd torsion / lateral bending	6	47,64	8,94	45%	91%	48%	90%
7	54,65	vertical pumping rear end	8	54,83	2,28	74%	100%	68%	99%
8	56,59	MFE and front end mode	8	54,83	2,28	38%	97%	42%	96%
9	57,00								
10	60,85	normal lower window mode	10	57,87	3,04	57%	95%	50%	94%
11	62,74	oscillating window mode	12	61,35	1,92	45%	98%	59%	96%
12	64,40	oscillating MFE mode	14	64,93	0,61	42%	101%	33%	96%
13	67,60	3rd lateral bending with MFE and roof parts	13	62,84	0,45	49%	93%	45%	92%
14	70,58	front end torsion mode	14	64,93	0,61	46%	92%	22%	95%

### Conclusion

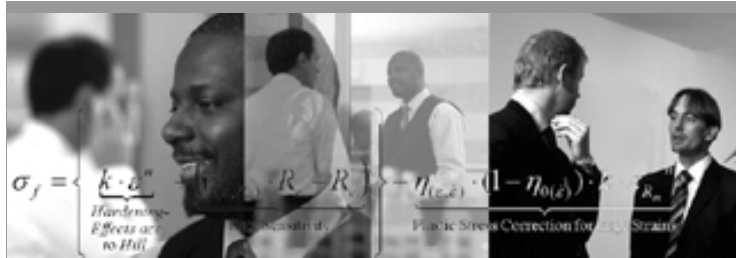
- Finer mesh results had a better starting point for the validation
- Global NVH modes can be analysed with refined meshes
- NVH Results seems to have small impact on mesh refinements
- Element resolution has an effect on the results
- Different element orientation give different results for coarser meshes
- Finer mesh is not so sensitive for different element orientation and changed boundary conditions (i.e. Rigid wall force)
  
- *Creation of finer meshes can be automated by **TEC|ODM!***

## Outlook

- Findings will be validated with more crash runs and NVH local modes
- Simulation results will be extended to further TEC|BENCH models (Renault Megane CC, Golf 5, etc.)







Thank you for your attention!  
Please ask some questions