

## The new Topcrunch benchmark data CAR 2 CAR for LS-DYNA<sup>®</sup> MPP971R5

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### Abstract

*CAR2CAR-ver10 has been used as benchmark data of Topcrunch since 2006. By the enhancement of tied contact in MPP971R5, this data do not work, since the slave nodes of constrained tied contact are released when master segments contain rigid body. CAR2CAR is revised to work for MPP971R5. This paper summarizes tied contact, then explain how to modify the data.*

### Introduction

Because of the enhancement of tied contact in MPP971R5, the Topcrunch benchmark data Car2car-ver10[1] does not work. In Car2Car, the solid elements are covered by null shell elements. Since the size of mesh between solid and shell are different, shell elements are tied to solid elements by constrained type tied contact \*CONTACT\_TIED\_NODES\_TO\_SURFACE. Some of solid elements are rigid bodies and by the constrained tied contact slave nodes can not be tied to rigid body. However, until MPP971R4 the slave nodes were tied to rigid body by constrained type contact, and the results looks good since gaps between slave nodes and master segment are zero for Car2car. After MPP971R5, the slave nodes are released when the nodes tied to master segment of rigid body by constrained type tied contact, and job terminated by error. New benchmark data Car2car-ver20 is modified to work after MPP971R5.

In this paper, first tied contacts are summarized. Then, the modification of car2car-ver20 is described. The modification of other than tied contact is also described.

Summary of tied contact[2]

| OFFSET            | projection | Tied method | Master segment<br>Rigid body | Orthogonality | TIED_NODES_TO_SURFACE<br>TIED_SURFACE_TO_SURFACE          | TIED_SHELL_EDGE_TO_SURFACE  |              |
|-------------------|------------|-------------|------------------------------|---------------|---|---|--------------|
|                   |            |             |                              |               | Degree of Freedom<br>X,Y,Z<br><br>Master Element<br>Solid | Degree of Freedom<br>X,Y,Z<br>Rx,RY,RZ<br><br>Master Element<br>Shell, beam |              |
| Without<br>OFFSET | Yes        | Constraint  | No                           | exact         |   |   |              |
| OFFSET            | No         | Constraint  | No                           | exact         | _CONSTRAINED_OFFSET<br>( careful use need )               | _CONSTRAINED_OFFSET   |              |
|                   |            | Penalty     | Incremental<br>Displacement  | Yes           | good  | _OFFSET<br>Option Card D<br>field7 TIEDID=1<br>( careful use need )         | _BEAM_OFFSET |
|                   |            |             | Total<br>Displacement        | Yes           | fair  | _OFFSET<br>(careful use need )  | _OFFSET      |

Projection : TIEDPRJ option of \*CONTROL\_CONTACT

TIEDPRJ=0(default ) slave nodes project to master surface, and eliminate gaps

=1 bypass projection, Gaps create rotational constraints which can substantially affect results

Orthogonality: exact=Tied constraint does not affect the rigid body motion

good=Tied constraint almost does not affect the rigid body motion

## Modification of tied contact

The modifications of tied contact are the following 3 types:

- A. Solid is rigid body

Shell nodes are constrained by \*CONSTRAINED\_EXTRA\_NODES\_SET

- B. Solid is deformable but some nodes are rigid constraint by

\*CONSTRAINED\_NODAL\_RIGID\_BODY

Shell nodes are tied by \*CONTACT\_TIED\_NODES\_TO\_SURFACE\_OFFSET with incremental displacement option

- C. Solid is deformable

Shell nodes are tied by \*CONTACT\_TIED\_NODES\_TO\_SURFACE

## Other Modification

The other modifications are

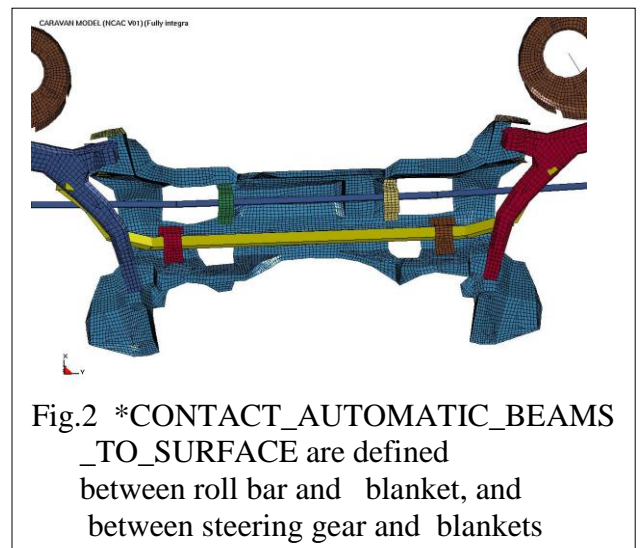
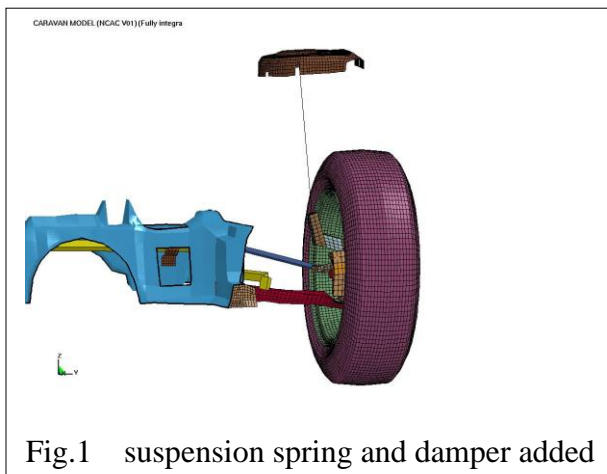
- A. Add spring and dumper elements for suspension as shown in Fig.1.

These elements are added by Caravan-V07-detail model[3].

- B. Among roll\_bar and steering gear and their blankets,

\*CONTACT\_AUTOMATIC\_BEAMS\_TO\_SURFACE are defined.

\*CONTACT\_AUTOMATIC\_BEAMS\_TO\_SURFACE is developed in MPP971R5.



### **Timing information**

|                        | 128Cores | 256Cores |
|------------------------|----------|----------|
| Total                  | 4h4m39s  | 3h1m19s  |
| Element calculation    | 8391s    | 4565s    |
| Contact calculation    | 4828s    | 4539s    |
| Rigid body calculation | 1358s    | 1625s    |

MPP971R6.1 rev 73149

Intel E5520 with infiniband

### **Discussions**

Penalty type tied contact without offset is missing in LS-DYNA. I would like to ask to add the capability of penalty type tied contact without offset.

The constrained type tied contact can not tie the segment with rigid body. The user have to select the slave nodes which is not tied, and these nodes are tied by penalty type tied contact. This process is inconvenient for users. I would like to propose the following tied contacts;

- \*CONTACT\_AUTOMATIC\_TIED\_NODES\_TO\_SURFACE,
- \*CONTACT\_AUTOMATIC\_TIED\_NODES\_TO\_SURFACE\_OFFSET,
- \*CONTACT\_AUTOMATIC\_TIED\_SHELL\_EDGE\_TO\_SURFACE, and
- \*CONTACT\_AUTOMATIC\_TIED\_SHELL\_EDGE\_TO\_SURFACE\_OFFSET.

By these automatic tied contacts,

1. Slave nodes tied to master segment by constrained tied contact
2. If master segment has rigid body and slave nodes do not tied by constrained tied contact.  
The slave nodes tied by penalty incremental displacement tied contact automatically.

### **References**

- [1] M.Makino, The performance of Large Car Model by MPP Version of LS-DYNA on Fujitsu Prime Power, 9<sup>th</sup> International LS-DYNA Users Conference, Computing/Code Technology(1), 2006
- [2] Livmore Software Technology Corp. LS-DYNA User's Manual
- [3] National Car Crash Center, <http://www.ncac.gwu.edu/vml/models.html>